Chapter 14 Fluids

01. The air inside pushes outward with a force given by p_iA , where p_i is the pressure inside the room and *A* is the area of the window. Similarly, the air on the outside pushes inward with a force given by p_0A , where p_0 is the pressure outside. The magnitude of the net force is $F = (p_i - p_0)A$. Since 1 atm = 1.013×10^5 Pa, F = (1.0 atm - 0.96 atm) (1.013×10^5 Pa/atm)(3.4 m)(2.1 m) = 2.9×10^4 N.

05. Let the volume of the expanded air sacs be V_a and that of the fish with its air sacs collapsed be *V*. Then $\rho_{\text{fish}} = m_{\text{fish}}/V = 1.08 \text{ g/cm}^3$ and $\rho_w = m_{\text{fish}}/(V+V_a) = 1.00 \text{ g/cm}^3$, where ρ_w is the density of the water. This implies $\rho_{\text{fish}}V = \rho_w(V+V_a)$ or $(V+V_a)/V = 1.08/1.00$, which gives $V_a/(V+V_a) = 7.4\%$.

07. (a) The pressure difference results in forces applied as shown in the figure. We consider a team of horses pulling to the right. To pull the sphere apart, the team must exert a force at least as great as the horizontal component of the total force determined by "summing" (actually, integrating) these force vectors. We consider a force vector at angle θ . Its leftward component is $\Delta p \cos \theta dA$, where dA is the area element for where the force is applied. We make use of the symmetry of the problem and let dA be that of a ring of constant θ on the surface. The radius of the ring is $r = R\sin\theta$, where R is the radius of the sphere. If the angular width of the ring is $d\theta$, in radians, then its width is $R d\theta$ and its area is $dA = 2\pi R^2 \sin\theta d\theta$. Thus the net horizontal component of the force of the air is given by

$$F = 2\pi R^2 \Delta p \int_0^{\pi/2} \sin \theta \cos \theta d\theta$$
$$= \pi R^2 \Delta p \sin^2 \theta |_0^{\pi/2} = \pi R^2 \Delta p.$$

(b) We use 1 atm = 1.01×10^5 Pa to show that $\Delta p = 0.90$ atm = 9.09×10^4 Pa. The sphere radius is R = 0.30 m, so $F_h = \pi (0.30 \text{ m})^2 (9.09 \times 10^4 \text{ Pa}) = 2.6 \times 10^4$ N. (c) One team of horses could be used if one half of the sphere is attached to a sturdy wall. The force of the wall on the sphere would balance the force of the horses.

11. The pressure p at the depth d of the hatch cover is $p_0 + \rho g d$, where ρ is the density of ocean water and p_0 is atmospheric pressure. The downward force of the water on the hatch cover is $(p_0+\rho g d)A$, where A is the area of the cover. If the air in the submarine is at atmospheric pressure then it exerts an upward force of p_0A . The minimum force that must be applied by the crew to open the cover has magnitude

$$F = (p_0 + \rho g d)A - p_0 A = \rho g dA$$

=
$$(1024 \text{ kg/m}^3)(9.8 \text{ m/s}^2)(100 \text{ m})(1.2 \text{ m})(0.60 \text{ m})$$

= $7.2 \times 10^5 \text{ N}$.

08. Note that 0.05 atm equals 5065 N/m^2 . Application of Eq. 14-7 with the notation in this prob. leads

to $d_{\text{max}} = 5065/\rho_{\text{liquid}g}$, with SI units understood. Thus the difference of this quantity between fresh water (998 kg/m³) and Dead Sea water (1500 kg/m³) is

$$d_{\max} = \frac{5065}{9.8} (\frac{1}{998} - \frac{1}{1500}) = 0.17 \text{ (m)}.$$

15. When the levels are the same the height of the liquid is $h = (h_1+h_2)/2$, where h_1 and h_2 are the original heights. Suppose h_1 is greater than h_2 . The final situation can then be achieved by taking liquid with volume $A(h_1-h)$ and mass $\rho A(h_1-h)$, in the first vessel, and lowering it a distance $h - h_2$. The work done by the force of gravity is $W = \rho A(h_1-h)g(h-h_2)$. We substitute $h = (h_1+h_2)/2$ to obtain $W = (\frac{1}{4})\rho g A(h_1-h_2)^2$

$$= (\frac{1}{4})\rho g A (h_1 - h_2)^2$$

$$= (\frac{1}{4})(1.30 \times 10^{3})(9.80)(4.00 \times 10^{-4})(1.56 - 0.854)^{2}$$

= 0.635 (J).

19. (a) At depth y the gauge pressure of the water is $p = \rho gy$, where ρ is the density of the water. We consider a horizontal strip of width W at depth y, with (vertical) thickness dy, across the dam. Its area is dA = Wdy and the force it exerts on the dam is $dF = pdA = \rho gyWdy$. The total force of the water on the dam is

$$F = \int_0^D \rho g W y dy = \frac{1}{2} \rho g W D^2 = \frac{1}{2} (1.00 \times 10^3 \text{ kg/m}^3)$$

(9.80 m/s²)(314 m)(35.0 m)² = 1.88 × 10⁹ N

(**b**) Again we consider the strip of water at depth y. Its moment arm for the torque it exerts about O is D - y so the torque it exerts is $d\tau = dF(D-y) = \rho gy W(D-y) dy$ and the total torque of the water is

$$\tau = \int_0^D \rho g y W(D - y) dy = \rho g W(\frac{1}{2} - \frac{1}{3}) D^3 = \frac{1}{6} \rho g W D^3$$
$$= \frac{1}{6} (1.00 \times 10^3) (9.80) (314) (35.0)^3 = 2.20 \times 10^{10} \text{ N} \cdot \text{m}$$

(c) We write $\tau = rF$, where *r* is the effective moment arm. Then,

$$r = \frac{\tau}{F} = \frac{\rho g W D^3 / 6}{\rho g W D^2 / 2} = \frac{D}{3} = \frac{35.0}{3} = 11.7 \text{ (m)}.$$

20. The gauge pressure you can produce is

$$p = -\rho gh = -(1000)(9.8)(4.0 \times 10^{-2})/(1.01 \times 10^{5})$$

= -3.9×10⁻³ (atm),

where the minus sign indicates that the pressure inside your lung is less than the outside pressure. 22. (a) According to Pascal's principle $F/A = f/a \Rightarrow$ F = (A/a)f. (b) We obtain

$$f = \frac{a}{A} F = \frac{(3.80 \text{ cm})^2}{(53.0 \text{ cm})^2} (20.0 \times 10^3 \text{ N}) = 103 \text{ N}.$$

The ratio of the squares of diameters is equivalent to the ratio of the areas. We also note that the area units cancel.

26. (a) The pressure (including the contribution from the atmosphere) at a depth of $h_{top} = L/2$ (corresponding *Chapter 14, HRW'04, NTOUcs960101*

to the top of the block) is

$$p_{\text{top}} = p_{\text{atm}} + \rho g h_{\text{top}} = [1.01 \times 10^5 + (1030)(9.8)(0.300)]$$

= 1.04×10⁵ (Pa),

where the unit Pa (Pascal) is equivalent to N/m². The force on the top surface (of area $A = L^2 = 0.36$ m²) is $F_{top} = p_{top}A = 3.75 \times 10^4$ N. (b) The pressure at a depth of $h_{bot} = 3L/2$ (that of the bottom of the block) is $p_{bot} = p_{atm} + \rho g h_{bot}$

= $[1.01 \times 10^5 + (1030)(9.8)(0.900)] = 1.10 \times 10^5$ (Pa), where we recall that the unit Pa (Pascal) is equivalent to N/m². The force on the bottom surface is F_{bot} = $p_{bot}A = 3.96 \times 10^4$ N. (c) Taking the difference $F_{bot}-F_{top}$ cancels the contribution from the atmosphere (including any numerical uncertainties associated with that value) and leads to

 $F_{\text{bot}} - F_{\text{top}} = \rho g(h_{\text{bot}} - h_{\text{top}})A = \rho g L^3 = 2.18 \times 10^3$ (N), which is to be expected on the basis of Archimedes' principle. Two other forces act on the block: an upward tension *T* and a downward pull of gravity *mg*. To remain stationary, the tension must be

$$T = mg - (F_{\text{bot}} - F_{\text{top}}) = (450)(9.80)$$
$$- 2.18 \times 10^3 = 2.23 \times 10^3 \text{ (N)}.$$

(d) This has already been noted in the previous part: $F_b = 2.18 \times 10^3$ N, and $T + F_b = mg$.

29. (a) Let *V* be the volume of the block. Then, the submerged volume is $V_s = 2V/3$. Since the block is floating, the weight of the displaced water is equal to the weight of the block, so $\rho_w V_s = \rho_b V$, where ρ_w is the density of water, and ρ_b is the density of the block. We substitute $V_s = 2V/3$ to obtain

 $\rho_b = 2\rho_w/3 = 2(1000 \text{ kg/m}^3)/3 = 6.7 \times 10^2 \text{ kg/m}^3.$ (b) If ρ_o is the density of the oil, then Archimedes' principle yields $\rho_o V_s = \rho_b V$. We substitute $V_s = 0.90V$ to obtain $\rho_o = \rho_b/0.90 = 7.4 \times 10^2 \text{ kg/m}^3.$

43. Suppose that a mass Δm of water is pumped in time Δt . The pump increases the potential energy of the water by Δmgh , where *h* is the vertical distance through which it is lifted, and increases its kinetic energy by $(\frac{1}{2})\Delta mv^2$, where *v* is its final speed. The work it does is $W = \Delta mgh + (\frac{1}{2})\Delta mv^2$ and its power is

$$P = \frac{\Delta W}{\Delta t} = \frac{\Delta m}{\Delta t} (gh + \frac{1}{2}v^2)$$

Now the rate of mass flow is $\Delta m/\Delta t = \rho_w A v$, where ρ_w is the density of water and A is the area of the hose. The area of the hose is $A = \pi r^2 = \pi (0.010 \text{ m})^2$ = $3.14 \times 10^{-4} \text{ m}^2$ and $\rho_w A v = (1000 \text{ kg/m}^3)(3.14 \times 10^{-4} \text{ m}^2)(5.00 \text{ m/s}) = 1.57 \text{ kg/s}$. Thus,

$$P = \rho A v [\rho g + \frac{1}{2} v^{2}] = (1.57 \text{ kg/s})[(9.80 \text{ m/s}^{2})(3.0 \text{ m}) + (\frac{1}{2})(5.0 \text{ m/s})^{2}] = 66 \text{ W}.$$

45. (a) We use the Eq. of continuity: $A_1v_1 = A_2v_2$. Here A_1 is the area of the pipe at the top and v_1 is the speed of the water there; A_2 is the area of the 此資料專為教學用請勿流傳-楊志信

pipe at the bottom and v_2 is the speed of the water there. Thus $v_2 = (A_1/A_2)v_1 = [(4.0 \text{ cm}^2)/(8.0 \text{ cm}^2)]$ (5.0 m/s) = 2.5 m/s.

(**b**) We use the Bernoulli Eq.: $p_1 + (\frac{1}{2})\rho v_1^2 + \rho g h_1 = p_2 + (\frac{1}{2})\rho v_2^2 + \rho g h_2$, where ρ is the density of water, h_1 is its initial altitude, and h_2 is its final altitude. Thus $p_2 = p_1 + \frac{1}{2}\rho (v_1^2 - v_2^2) + \rho g (h_1 - h_2)$

$$= 1.5 \times 10^5 + \frac{1}{2} (1.0 \times 10^3) (5.0^2 - 2.5^2)$$

+ $(1.0 \times 10^3)(9.8)(10) = 2.6 \times 10^5$ (Pa).

54. (a) The volume of water (during 10 minutes) is $V = (v_1 t)A_1 = (15 \text{ m/s})(10 \text{ min})(60 \text{ s/min})(\pi/4)$ (0.03 m)² = 6.4 m³. (b) The speed in the left section of pipe is

$$v_2 = v_1(\frac{A_1}{A_2}) = v_1(\frac{d_1}{d_2})^2 = (15)(\frac{3.0}{5.0})^2 = 5.4 \text{ (m/s)}.$$

(c) Since $p_1 + (\frac{1}{2})\rho v_1^2 + \rho g h_1 = p_2 + (\frac{1}{2})\rho v_2^2 + \rho g h_2$ and $h_1 = h_2$, $p_1 = p_0$, which is the atmospheric pressure,

$$p_2 = p_0 + \frac{1}{2}\rho(v_1^2 - v_2^2) = 1.01 \times 10^5 + \frac{1}{2}(1.0 \times 10^3)$$

 $(15^2-5.4^2) = 1.99 \times 10^5$ (Pa) = 1.97 (atm). Thus the gauge pressure is (1.97 atm - 1.00 atm) = 0.97 atm = 9.8 \times 10^4 Pa.

55. (a) Since S.P. 14-8 deals with a similar situation, we use the final Eq. (labeled "Answer") from it: $v = (2gh)^{1/2} \Rightarrow v = v_0$ for the projectile motion. The stream of water emerges horizontally ($\theta_0 = 0^\circ$ in the notation of Chapt. 4), and setting $y - y_0 = -(H-h)$ in Eq. 4-22, we obtain the "time-of-flight"

$$t = \sqrt{\frac{-2(H-h)}{-g}} = \sqrt{\frac{2}{g}(H-h)}$$
.

Using this in Eq. 4-21, where $x_0 = 0$ by choice of coordinate origin, we find

$$x = v_0 t = \sqrt{2gh} \sqrt{2(H-h)/g} = 2\sqrt{h(H-h)}$$
$$= 2\sqrt{10(40-10)} = 35 \text{ (m)}.$$

(b) The result of part (a) (which, when squared, reads $x^2 = 4h(H-h)$) is a quadratic Eq. for *h* once *x* and *H* are specified. Two solutions for *h* are therefore mathematically possible, but are they both physically possible? For instance, are both solutions positive and less than *H*? We employ the quadratic formula:

$$h^{2} - Hh + \frac{1}{4}x^{2} = 0 \Longrightarrow h = \frac{1}{2} (H \pm \sqrt{H^{2} - x^{2}}),$$

which permits us to see that both roots are physically possible, so long as $x \le H$. Labeling the larger root h_1 (where the plus sign is chosen) and the smaller root as h_2 (where the minus sign is chosen), then we note that their sum is simply

$$h_1 + h_2 = \frac{1}{2}(H + \sqrt{H^2 - x^2}) + \frac{1}{2}(H - \sqrt{H^2 - x^2}) = H.$$

Thus, one root is related to the other (generically

labeled h' and h) by h' = H - h. Its numerical value is h' = 40 cm - 10 cm = 30 cm. (c) We wish to maximize the function $f = x^2 = 4h(H-h)$. We differentiate with respect to h and set equal to zero to obtain

$$df/dh = 4H - 8h = 0 \implies h = H/2,$$

or h = (40 cm)/2 = 20 cm, as the depth from which an emerging stream of water will travel the maximum horizontal distance.

69. (a) We consider a point D on the surface of the liquid in the container, in the same tube of flow with points A, B and C. Applying Bernoulli's Eq. to points D and C, we obtain

$$p_D + \frac{1}{2}\rho v_D^2 + \rho g h_D = p_C + \frac{1}{2}\rho v_C^2 + \rho g h_C$$
,

which leads to

$$v_c = \sqrt{\frac{2(p_D - p_C)}{\rho} + 2g(h_D - h_C) + v_D^2} \approx \sqrt{2g(d + h_2)}.$$

where in the last step we set $p_D = p_C = p_{air}$ and v_D/v_C ≈ 0 . Plugging in the values, we obtain

$$v_c = \sqrt{2(9.8)(0.40 + 0.12)} = 3.2 \text{ (m/s)}.$$

(**b**) We now consider points *B* and *C*: $p_B + \frac{1}{2}\rho v_B^2 + \rho g h_B = p_C + \frac{1}{2}\rho v_C^2 + \rho g h_C$.

Since $v_B = v_C$ by Eq. of continuity, and $p_C = p_{air}$, Bernoulli's Eq. becomes

$$p_B = p_C + \rho g(h_C - h_B) = p_{air} - \rho g(h_1 + h_2 + d)$$

= 1.0×10⁵ - (1.0×10³)(9.8)(0.25+0.40+0.12)
= 9.2×10⁴ (Pa).

(c) Since $p_B \ge 0$, we must let $p_{air} - \rho g(h_1 + d + h_2) \ge$ 0, which yields

$$h_1 \le h_{1,\max} = \frac{p_{\text{air}}}{\rho} - d - h_1 \le \frac{p_{\text{air}}}{\rho} = 10.3 \text{ m}$$

80. The absolute pressure is

 $p = p_0 + \rho g h = 1.01 \times 10^3 \text{ Pa} + (1.03 \times 10^3 \text{ kg/m}^3)$ $(9.80 \text{ m/s}^2)(150 \text{ m}) = 1.62 \times 10^4 \text{ Pa}.$

84. (a) Using Eq. 14-10, we have $p_g = \rho g h = 1.21 \times$ 10^7 Pa. (b) By definition, $p = p_g + p_{atm} = 1.22 \times 10^7$ Pa. (c) We interpret the question as asking for the total force compressing the sphere's surface, and we multiply the pressure by total area: $p(4\pi r^2) = 3.82$ $\times 10^5$ N. (d) The (upward) buoyant force exerted on the sphere by the seawater is $F_b = \rho_w g V$, where V = $(4/3)\pi r^3$. Therefore, $F_b = 5.26$ N. (e) Newton's second law applied to the sphere (of mass m = 7.00 kg) yields $F_b - mg = ma$, which results in a = -9.04 m/s^2 , which means the acceleration vector has a magnitude of 9.04 m/s^2 . (f) The direction is downward.

92. (a) We assume that the top surface of the slab is at the surface of the water and that the automobile is at the center of the ice surface. Let *M* be the mass of the automobile, ρ_i be the density of ice, and ρ_w be the density of water. Suppose the ice slab has

area A and thickness h. Since the volume of ice is Ah, the downward force of gravity on the automobile and ice is $(M + \rho_i Ah)g$. The buoyant force of the water is $\rho_w Ahg$, so the condition of equilibrium is $(M + \rho_i Ah)g - \rho_w Ahg = 0$ and

$$A = \frac{M}{(\rho_w - \rho_i)h} = \frac{1100}{(998 - 917)(0.30)} = 45 \text{ (m}^2\text{)}.$$

These density values are found in Table 14-1 of the text. (b) It does matter where the car is placed since the ice tilts if the automobile is not at the center of its surface.

91.[•] Equilibrium of forces (on the floating body) is expressed as $F_b = m_{\text{body}}g$ $\Rightarrow \rho_{\text{liquid}} g V_{\text{submerged}} =$ $\rho_{\rm body} g V_{\rm total}$, which leads to

$$\frac{V_{\text{submerged}}}{V_{\text{total}}} = \frac{\rho_{\text{body}}}{\rho_{\text{liquid}}} \,.$$

We are told (indirectly) that two-thirds of the body is below the surface, so the fraction above is 2/3. Thus, with $\rho_{\text{body}} = 0.98 \,\text{g/cm}^3$, we find $\rho_{\text{liquid}} \approx 1.5 \,\text{g/cm}^3$ certainly much more dense than normal seawater (the Dead Sea is about seven times saltier than the ocean due to the high evaporation rate and low rainfall in that region).

59.* (a) The continuity Eq. yields Av = aV, and Bernoulli's Eq. yields $\Delta p + (1/2)\rho v^2 = (1/2)\rho V^2$, where $\Delta p = p_1 - p_2$. The first Eq. gives V = (A/a)v. We use this to substitute for V in the second Eq., and obtain $\Delta p + (\frac{1}{2})\rho v^2 = (\frac{1}{2})\rho (A/a)^2 V^2$. We solve for v. The result is

$$v = \sqrt{\frac{2\Delta p}{\rho[(A^2 / a^2) - 1]}} = \sqrt{\frac{2a^2 \Delta p}{\rho(A^2 - 1)}}.$$

(**b**) We substitute values to obtain

$$v = \sqrt{\frac{2(32 \times 10^{-4})(55 \times 10^{3} - 41 \times 10^{3})}{(1000)[(64 \times 10^{-4})^{2} - (32 \times 10^{-4})^{2}]}} = 3.06 \text{ (m/s)}.$$

Consequently, the flow rate is $Av = (64 \times 10^{-4} \text{ m}^2)$ $(3.06 \text{ m/s}) = 2.0 \times 10^{-2} \text{ m}^3/\text{s}$.

61.* (a) Bernoulli's Eq. gives $p_A = p_B + (\frac{1}{2})\rho_{air}v^2$, But $\Delta p = p_A - p_B = \rho g h$ in order to balance the pressure in the two arms of the U-tube. Thus $\rho gh = (\frac{1}{2})\rho_{air}v^2$, $v = \sqrt{\frac{2\rho gh}{\rho_{air}}}.$ or

(b) The plane's speed relative to the air is

$$v = \sqrt{\frac{2(810)(9.8)(0.260)}{1.03}} = 63.3$$
 (m/s).

◆靜流體壓力(v=0):壓力 p + 重力位能密度pgy = 常數 ◆連續方程式:體流率 Av =常數 ◆柏努 利方程式:壓力 p+ 動能密度(1/2) pv2+ 重力位能 密度pgy = 常數 ◆亞基米得原理:一部份或完全 浸於流體內的物體,被一等於其排開流體重量之 力所浮升,即**浮力** $F_B = \rho_f Vg$ 。

Chapter 14, HRW'04, NTOUcs960101

重點整理-第 14 章 流體 什麼造成抓地效應?又為什麼它會消失?

密度	任-	-物質的密度	o定義爲每單	位體積的質量:	:
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 $\rho = \Delta m / \Delta V. \tag{14-1}$

通常物質試樣遠大於原子尺度,14-1式可寫為

$$\rho = m/V. \tag{14-2}$$

流**體壓力**流體為可流動的物質;其因無法承受切應力,形狀隨容器形狀而改變;然而它能施加一 垂直於其表面的力,該力可藉壓力*p*以描述:

$$p = \Delta F / \Delta A, \qquad (14-3)$$

其中ΔF 為作用於面積ΔA 之表面元素的力。假如 力均匀施於平坦表面上,則 14-3 式可寫為

$$p = F/A. \tag{14-4}$$

於流體中某特定點由流體壓力產生的力,其大小 於各方向均相等。**計示壓力**為在某點真實的壓力 (或絕對壓力)與大氣壓力之差。

壓力隨高度及深度變化在靜止流體內壓力隨鉛 直位置 y 而改變,對 y 向上取正而言,

$$p_2 = p_1 + \rho g(y_1 - y_2). \tag{14-7}$$

流體內所有同一高度的點之壓力均相等。假如 h 為流體試樣在壓力為 po 某參考高度之下的深度, 14-7 式變為

 $p = p_0 + \rho gh, \qquad (14-8)$ 式中p爲試樣內的壓力。

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- J3.**大氣中的狂暴舞者—龍捲風**,李健成,科學發展 378 期(9306) 52。
- J4.旋,還是不旋一變化球問題,李志揚、陳政宏,科 學發展 388 期(9404) 54。
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- J5.Q&A.高爾夫球上的小洞有什麼作用?科學人 2004 年5月號。
- J6.Q&A.正常血壓值為什麼應小於 120/ 80?為什麼不 會隨身高改變?科學人 2005 年 5 月。
- J8.**耀眼的六百年—鄭和與〈鄭和航海圖〉**, 龍村倪, 科學發展 379 期(9307) 42。

此資料專為教學用請勿流傳-楊志信 帕斯卡原理 施於密閉容器中流體之壓力改變 量,必不滅地傳遞至流體每一部份與容器壁上。 亞基米德原理 當物體完全或部份浸於流體中,週 遭的流體作用於物體造成一浮力 \vec{F}_b 。此力垂直向 上且其大小為 $F_b = m_fg$, (12-24) 式中 m_f 為物體所排開流體的質量(亦即被物體排 擠出的流體)。

當物體浮於流體上,作用於物體的(向上)浮力 大小 F_b等於作用於物體的(向下)重力大小 F_g。浮 力作用的物體之視重與真實重量的關係為

weight_{app} = weight – F_b (14-19) **理想流體**為不可壓縮且無黏滯 性,並且其流動爲穩定的且無旋的。流錄爲各流 體質點所依循的路徑。流動管爲一整束流線。於 任一流動管中流動遵循連續方程式:

 $R_v = Av = 常數,$ (12-25) 其中 R_v 爲體積流率, A 爲流管中任意點的截面 積,而 v 爲該點流體速率。質量流率 R_m 爲

 $R_m = \rho R_v = \rho A v = 常數.$ (14-25) 柏努利方程式 應用力學能守恆原理於理想流體 之流動導出沿著任意流動管之柏努利方程式:

$$p + \frac{1}{2}\rho v^2 + \rho gy = 常數$$
 (14-29)

(如發現錯誤煩請告知 jyang@mail.ntou.edu.tw, Thanks.) Archimedes 亞基米得; Bernoulli 柏努利; Pascal 帕斯卡; Rayleigh 瑞立; Torricelli 托里切利; fluid 流體; flow 流 動; tube of flow 流動管; laminar flow;層流; equation of continuity 連續方程式; mass/volume flow rate 質/體流 率; Mercury barometer 水銀氣壓計; open-tube manometer 開管壓力計; pressure 壓力; absolute/gauge pressure 絕對/計示壓力; systolic/diastolic pressure 收縮/舒張壓; mm Hg 毫米汞柱; dam 水壩; hydraulic lever 液壓槓桿; flap 副翼; siphon 虹吸管; wing 尾翼,擾流板; buoyant force 浮力; density 密度; desperado 歹徒; dye 染料; Ethanol 乙醇; Heimlich maneuver 海姆利胥急救法; negative lift 負升力; nonviscous 無黏滯的; pilot tube 領 示管; irrotational 無旋的; scuba 水肺; streamline 流線; torpedo 魚雷; turbulent 急,湍流的; venturi meter 文士里 計量計;

•備忘錄•